Load-holding valves type LHDV

with special oscillation dampening, zero leakage

1. General

These valves are pressure valves according to the Industrial Standard DIN ISO 1219-1. They prevent pulling or pushing loads from accelerating uncontrollably during movements in load direction, or from proceeding with higher speed than intended i.e. determined by the inflowing oil on the pump's side. Consequently, these devices prevent a collapse or eventual rupture of the oil column. The main application for load-holding valves is with hydraulic lifting-, pivoting-, turning- or similar constructions which utilize double acting consumers (hydraulic cylinders, hydraulic motors).

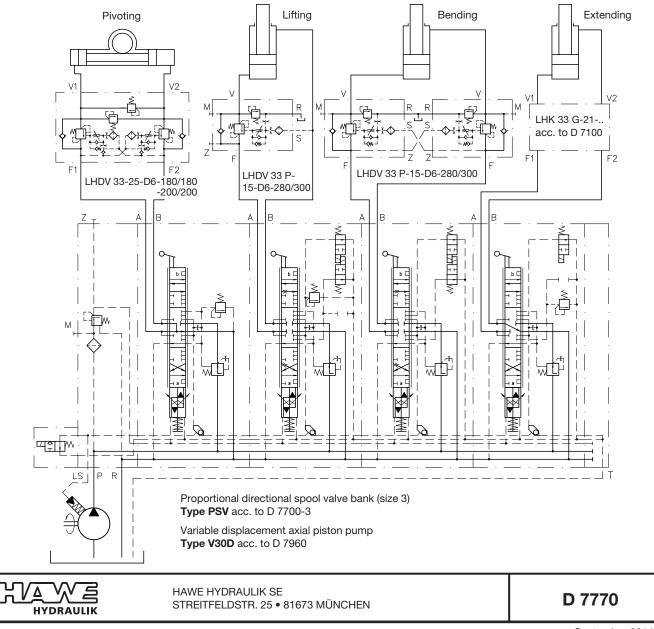
This is accomplished by throttling of the return flow from the corresponding consumer. The load-holding valves generates a flow resistance, which is always a little bit higher than the pressure created from the load. This back pressure is only generated under negative load conditions. But the valve will be fully opened, enabling free flow (return) if the load is positive, i.e. the load acts against the direction of the motion.

The throttle device is self-adjusting and therefore adapts continuously to any alternation of the load condition. This is achieved by an equilibrium of forces between the outflow and inflow (from the actuated consumer) acting on the functional valve elements on the one side, and the valve spring acting on the other side.

The valves version LHDV are especially designed for those applications, which, due to their own elasticity, tend heavily towards pendulum oscillations. The load-holding valves are most advantageous when utilized in conjunction with prop. directional spool valve banks, functioning according to the Load-Sensing-Principle which do have 2-way inflow control valves in each valve bank section. Consequently, they should be installed in the corresponding lines between consumer and directional spool valve.

As a self-contained unit, the LHDV-valve permits the specific intervention into the oscillating circle, as it is created by hydraulic cylinders with attached load, the flow control valves of the directional spool valves, or the pressure/flow regulator of variable displacement pump. Its dampening abilities are significantly more adaptable and their effect more accurately adjustable than would be possible with common measurements, e.g. through modification (distortion) of the characteristic curve of the flow control valves installed in prop. directional spool valves.

The fluctuating load pressure influences the motion of the control device which varies the throttle area. But its response is slightly delayed, slowed and weakened by a combination of especially designed damping elements. This will successfully intercept the pendulum motions being evoked, which are induced by starting, stopping, or sudden transitions from full speed to crawl speed. They will be eventually suppressed in their developing stage, by letting them fade away quickly. For a detailed functional description and notes for customizing the damping, especially for critical conditions, see B 7770.



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2.3

2. Available versions, main data

Order

examples:

Desired pressure setting (bar) within the permissible pressure range, acc. to sect. 3

For correct positioning of pressure figures for the load holding valve, and eventual shock valve see following examples.

- Table 3: Orifice combinations (orifice D1 - without coding = 0.5 mm)

Coding	Orifice 2					
	4	5	6 (Standard)	7	8	0
Ø (mm)	0.4	0.5	0.6	0.7	0.8	0 (no hole)
Release ratio ¹) The actual re		1	1: 2.9			1: 8.2 ¹)

Table 2: Flow adaptation

LHDV 33 P - 15 - B 6 - 300/320

Adjustable pressure ranges	(50) 350	А	в	с	D	E	²) With positively acting load, i.e. during lifting,
pressure ranges p _{max} (bar)	351 420	L	м	N	Р	R	one can expect a Δp
Max. flow V→F Q _{max} approx. (Ip valve fully opene		this a press	60 Iso the 2 pplies a ure V \rightarrow comple	lso to tł F with	ne flow positive	/ back	of approx. 50 bar with the max. permissible flow rates. This pres- sure has to be added to the load pressure.

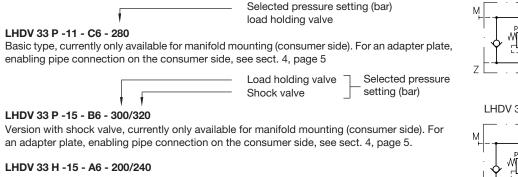
 Table 1: Basic type, size, and additional elements

Flow pattern symbols, for illustration see sect. 2.1			Single valves for constant load directions	Double valv Without additional elements		ttle valves fo	ad direction or the pressure signal port X(T) ass check ort X With suction port T (volume balance)
	Standard		11 ³)	21	21W	21WD	
		With unpressurized control piston		21L	21WL		
Basic type	With add.		15 ³)	25	25W	25WD	25WDN
Design	shock- valves	With unpressurized control piston		25L	25WL	25WDL	25WDNL
LHDV 33 -	Pipe conne	ection ⁴)		•	•	٠	•
LHDV 33 P -	Manif. mou	anif. mounting, cons. side • 3) Port Z is unplugged ex-works (see following flow p		rks (see following flow pat-			
LHDV 33 H -	Banjo bolt mounting, consumer	M22x1.5 metric fine thread DIN 13 T6	•	by a t	apped pl		ocked later, if not required -G 1/4 A-St with seal ring
LHDV 33 H 1/2 -	side	G 1/2 A ⁴)	•	⁴) ISO 228/1 (BSPP)			

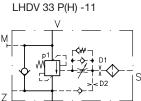
2.1 Additional order examples with corresponding flow pattern symbols Single valves for always constant load direction

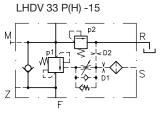
Release for direction V \rightarrow F during lowering of the load via an external control line at port S from the other side (inflow consumer line).

Order examples of available versions:



Version with shock valve, mounted by banjo bolt H = M 22x1.5 or H 1/2 - G 1/2 A (consumer side). It may be installed in any angel concentric around the V-port. A centering predestal is required at the mounting area, see dimensional drawings sect. 4





Double valve for alternating load directions

The release of the respective reflow side V1 \rightarrow F1 or V2 \rightarrow F2 takes place via internal control oil ducts. No external control pipes are required.

Oreder examples of available versions:



LHDV 33 - 21 - A6 - 240/180

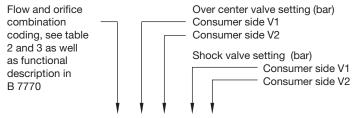
Basic version for all applications, where no high pressure peaks or sudden user stops (shock pressure) are expected.

LHDV 33 - 21L - A6 - 240/180

Like the above basic version, but with additional port for leakage oil (see also notes in sect. 5.2).

LHDV 33 - 21W(WD) - A6 - 240/180

Like the above basic version, but with additional shuttle valve (see also description of type LHDV 33 - 25W(WD))

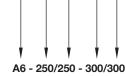


LHDV 33 - 25 - D5 - 220/220 - 260/260

LHDV 33 - 25W -

Basic version with shock valves e.g. for consumers with a piston area ratio of 1:1.

Flow pattern symbol for version LHDV 21-25L with additional oil leakage port, similar to LHDV 33 - 21L...



Like the basic version 25, but with additional shuttle valve, e.g. for brakes with hydraulic release (port X). Preferably used for hydraulic motors. Flow pattern symbol for version LHDV 33-25WL with additional leakage port, similar to LHDV 33 - 21L...



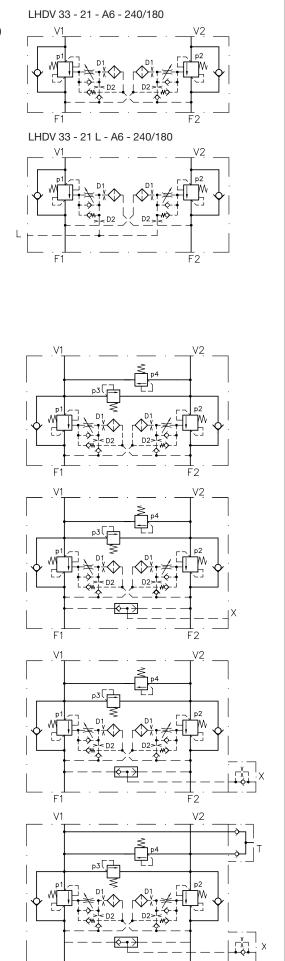
Like version 25 W, but with additional by-pass check valve type BC1-40 E acc. to D 6969 B mounted at port X (intended to prevent sudden kicking-in of the brake).

Flow pattern symbol for version LHDV 33-25 WDL with additional leakage port, similar to LHDV 33 - 21L...





Like version 25 WD, but with additional suction valve No. 7770 040 intended for hydraulic motors to balance volumes, altered by leakage. Symbol for version LHDV 33 -25 WDNL with additional leakage port, similar to LHDV 33 - 21L...



F2

3. Further characteristic data

	Designation	Load holding valve (over center valve), with hydraulic relea	ase and by-pa	iss check valve				
	Valve design	Load holding valve: cone seated piston valve By-pass check valve: disk seated valve						
	Installed position	Amy						
	Ports	F, F1, F2, V, V1, V2 and R Main ports M, S, Z Control- and probing ports de	epending on v	version				
	Mass (weight) approx.	Type LHDV 33 P-11 = 1.3 kg LHDV 33-21(21W) LHDV 33 P-15 = 1.8 kg 1) LHDV 33-21L (21WL) LHDV 33 H-11 = 1.7 kg LHDV 33-21WD LHDV 33 H-15 = 2.2 kg LHDV 33-25 (L, W, WI 1) Corresponding connection block No. 7770 024 = 0.4 kg	= 3.6 kg L) = 3.9 kg	LHDV 33-25WD = 4. LHDV 33-25WDN = 4. LHDV 33-25WDNL = 4.				
	-	,	-					
	Flow direction	Working direction (load holding function) V \rightarrow F, V1 \rightarrow F1 or free flow F \rightarrow V, F1 \rightarrow V1, F2 \rightarrow V2	V2→F2					
	Release ratio	approx. 1:8.2 with closed valve (geometrical ratio) approx. 1:1.2 to 1:6.4 with open (unlocked) valve, dependir see sect. 2, table 3	ng on the orific	e diameter ratio,				
	Pressure adjustment	A pressure gauge should be used whenever the pressure setting is adjusted or altered! The g figures for pressure alternation per rotation or per mm adjustment travel of the perforated disc w the connector F (F1 and F2) are only a rough guide line for approximately achieving the desired set (start of operation). The setting should be at least 10% above the max. expected load pressure.						
		Alternation of pressure approx .:	per turn	per mm approx.				
		Load holding valvepressure range50 250 barpressure range251 350 barpressure range351 420 bar	45 bar 50 bar 62 bar	25 bar 27.5 bar 34 bar				
		Shock valve pressure range 50 450 bar	106 bar	80 bar				
	pump gauge		any Perf Allar a/f 6	 and must be loosened pripressure adjustments orated disc can be rotated h Key - Load holding valve - Shock valve - Pressure increases 				
	should be circulating via	e is necessary with test rigs using a motor pump! The pump open throttle valve, then close the throttle valve slowly until nding (avoid larger flow since the valve might squeal).	🙂 Reti	I = Pressure decreases ghten the grub screw ormed adjustment				
	Pressure fluid	Hydraulic oil according to DIN 51 524, table 1 to 3; ISO VG 10 to 68 according to DIN 51 519 range of viscosity: min. approx. 4; max. approx. 1500 mm ² /sec; optimum range: app 10500 mm ² /sec. Also usable for biodegradable pressure fluids of the type HEPG (Polyalcylengly and HEES (synthetic ester) at operating temperatures < +70°C. Ambient: approx40 +80°C Fluid: -25 +80°C, but pay attention to viscosity Starting temperature down to -40°C admissible (watch starting viscosity!), when the operat temperature during following operation is at least 20 K higher. Biological degradable pressure fluids: Observe manufacturer's specifications. Considering compatibility with seal material not over < +70°C.						
	Temperatures							
	Δp -Q-curves	The curves (reference values) for $V \rightarrow F$ are valid for the full	lly opened (rel	eased) valve				
10	Pressure range A	$\begin{array}{c c} & & & & \\ & & & & \\ & & & \\ &$	D	Pres. range D, E				
60 40 20 0		20 F V G Range B, C	D	Pres. range D, E 10 15 20 Flow Q (lpm)				

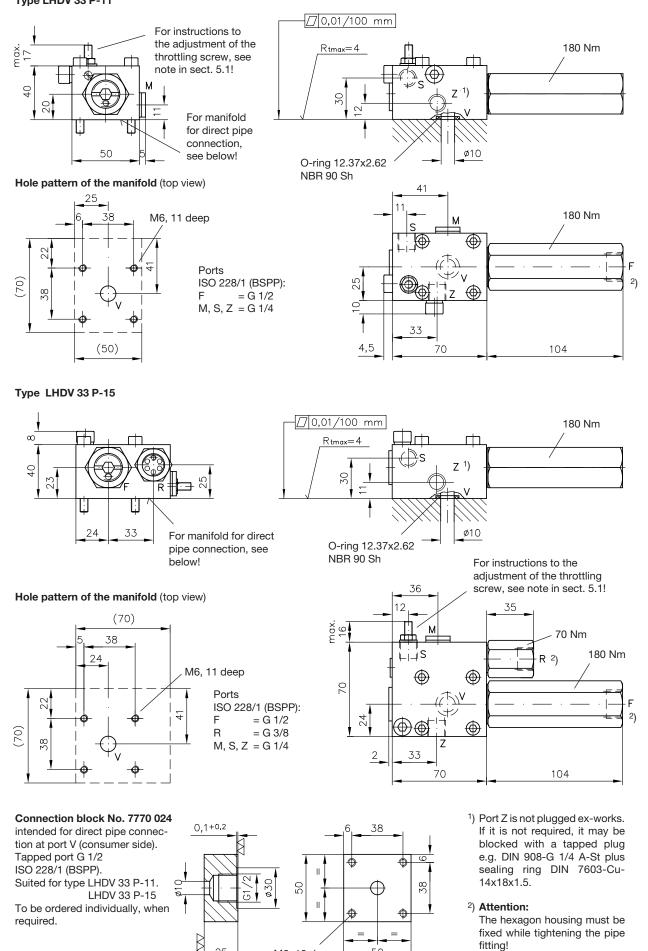
The double valves, flow pattern symbol 21... and 25..., cannot be utilized with directional valves, which show the flow characteristics of a differential circuit in one position, e.g. coding C in pamphlet D 5700. Single valves, flow pattern symbols 11 or 15, must not be connected to the rod side of hydraulic cylinders.



4. **Dimensions** All dimensions in mm and subject to change without further notice!

For the accessibility of the adjustable damping devices, see functional description B 7770.

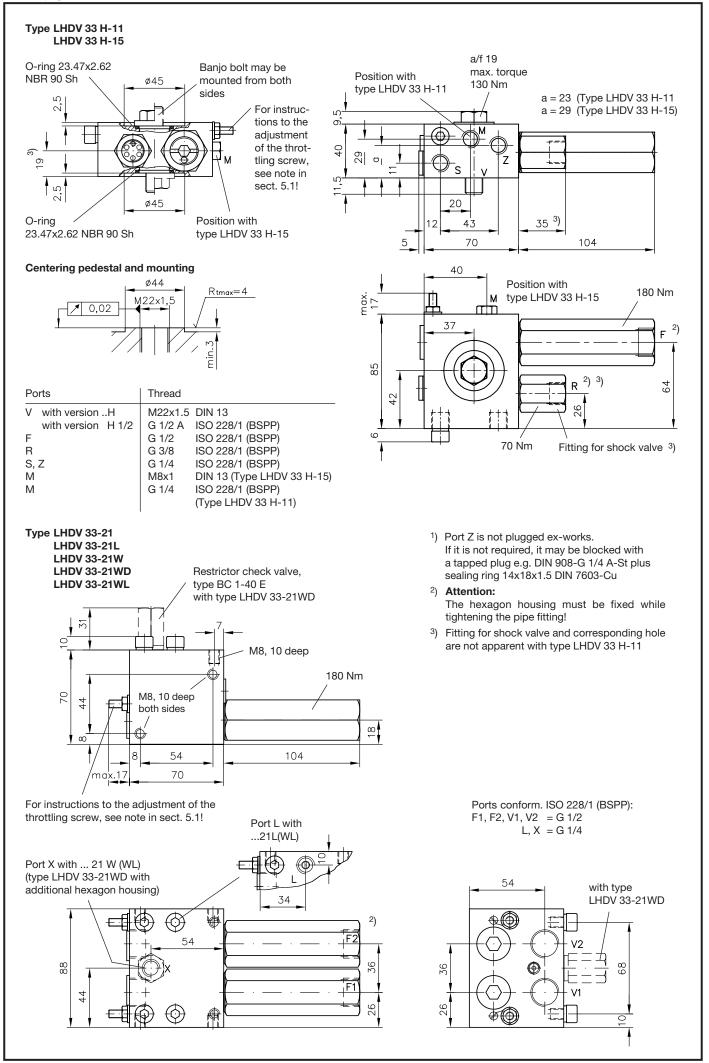
Type LHDV 33 P-11

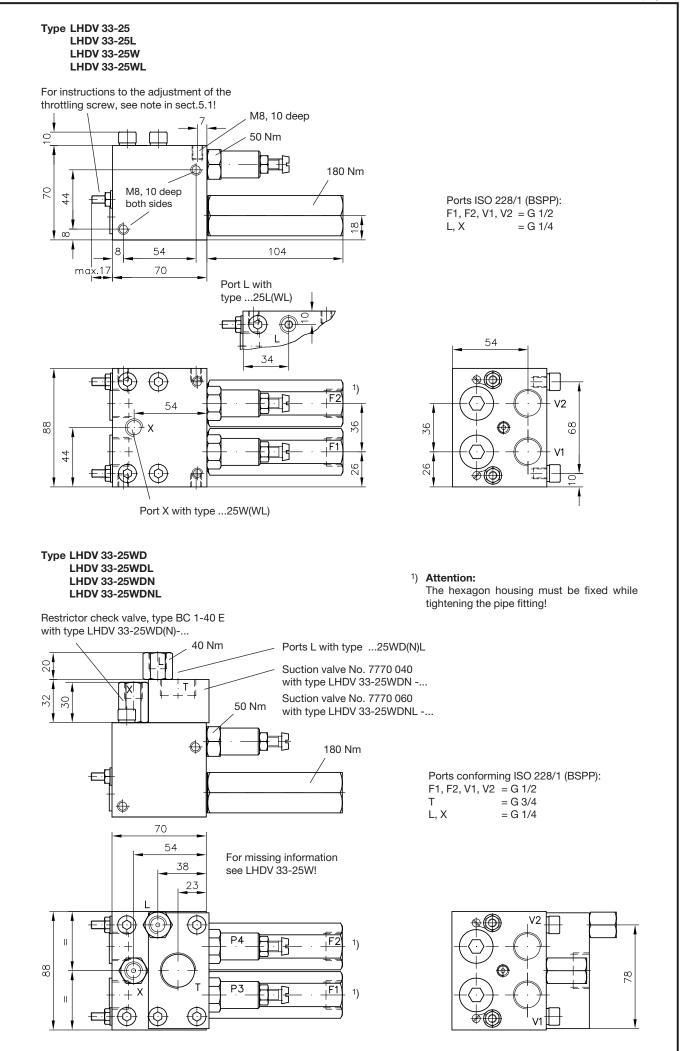


50

M6, 10 deep

25





5. Appendix

5.1 Dampening throttles

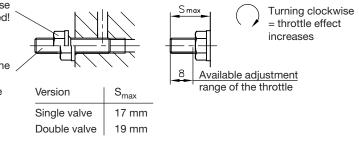
The dampening behavior may be extensively adapted within the adjusting range. This can be performed on site. It is recommended to include the following note and the schematic drawing into the operating manual or the operating instructions of the equipment.

The lock nut a/f 10 (Seal-Lock nut) needs to be loosened sufficiently prior to adjusting the throttling screw, otherwise the vulcanized sealing gasket of the thread will be damaged!

Throttle screw

(grub screw ISO 4026 M6^{4h} x 30-8.8-A2K)

Attention: Do not unscrew the throttle screw above the S_{max} (as is illustrated in the adjacent figure)! Due to construction restrictions it cannot be anchored on the inside of the equipment.



5.2 Release pressure p_{in} on the inflow side

The required pressure at the pump p_{in} to transfer the load against the load holding valve located down stream (direction V \rightarrow F) can't be exactly predicted. It depends on the following parameter: Piston cross section area ratio A_{in} : A_{out} of the hydraulic cylinder, the internal operation area ratio of the load holding valve (release ratio acc. to sect. 3), the existing load pressure and the flow resistance $\Delta p_{F(R)}$ of all additional throttling locations downstream back to the tank e.g. reflow pipe, directional valves (in the example $A \rightarrow R$).

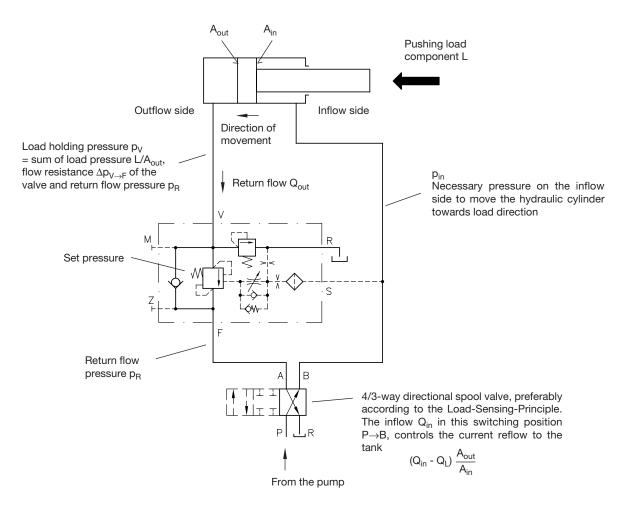
The setting of an additional shock valve installed in the feeding pipe of the consumer has to be adjusted high enough, over the setting of the main pressure relieve valve, that it can overcome the highest release pressure (no load situation).

Rough guiding figures suitable for a max. set pressure of 370 or 250 bar and max. flow dep. on valve coding, see sect. 2:

 p_{in} max. \approx 130...170 bar at 370 bar set pressure \approx 100...140 bar at 250 bar set pressure

with a piston cross section area ratio A_{in} : A_{out} of about 2...0.5 for the hydraulic cylinder. The return flow resistance can increase these standard guiding figures by about (1.1...3.5) x $\Delta p_{F(R)}$ depending on the release ratio.

A readjustment of the pressure limiting valve is possible on site when required.



Important note:

The additional leakage port of the double valves acc. to sect. 2, page 3 (e.g. LHDV 33-21L -...) reduces the influence of the return flow resistance back to the tank. An additional advantage is in the possibility that this leakage pipe, in the case of an emergency, can be shut-off with a hand pump.